

1078
B.E. (Mechanical Engineering)
Third Semester
MEC-303: Theory of Machines – I

Time allowed: 3 Hours

Max. Marks: 50

NOTE: Attempt five questions in all, including Question No. 1 which is compulsory and selecting two questions from each Part.

x-x-x

- Q1a) Classify different types of links.
- b) What is Coriolis component of acceleration
- c) Define transmission angle.
- d) Define coefficient of fluctuation of energy
- e) What is the effect of slip and creep on velocity ratio of belt drive (10)

Part-A

- Q2a) Sketch and explain any one inversion of double slider crank chain. (2)
- b) What is Grubler criterion. (2)
- c) State and prove Kennedy Theorem (3)
- d) In a four bar chain ABCD, AD is fixed and is 15 cm long. The crank AB is 4 cm long and rotates at 120 rpm clockwise, while the link CD is 8 cm long and oscillates about D. BC and AD are of equal length. Find the angular velocity of link CD when angle BAD = 60°. (3)

- Q3 a) Write Freudenstein's equation. (2)
- b) Synthesize a four bar linkage using Freudenstein's equation to generate the function $y=x^{1.5}$ for the interval $1 \leq x \leq 4$. The input crank is to start from $\theta_2=30^\circ$ and is to have a range of 90° . Take three accuracy points. Take output crank angle from 0° to 90° . (8)

- Q4a) The turning moment curve for an engine is represented by the equation:
 $T=2000 + 950 \sin 2\theta - 570 \cos 2\theta$ kg-m where θ is the angle moved by the crank from inner dead centre. If the resisting torque is constant find:
 - A) Power developed by the engine
 - B) Moment of inertia of flywheel in kg-m^2 , if total fluctuation of speed is not to exceed one percent of mean speed which is 180 rpm and
 - C) Angular acceleration of flywheel when the crank has turned through 45° from inner dead centre. (6)
- b) Explain the procedure of static force analysis of slider crank mechanism. (4)

Part-B

- Q5a) Prove that maximum efficiency of screw jack is given by:
 $\eta_{\max} = \frac{1 - \sin \Phi}{1 + \sin \Phi}$; where Φ = friction angle. (5)
- b) A leather faced conical friction clutch has a cone angle of 30° . The intensity of normal pressure between the contact surface is not to exceed $6 \times 10^4 \text{ N/m}^2$ and the breadth of conical surface is not to be greater than $1/3$ of mean radius. Find the dimensions of contact surface if $\mu=0.20$ and the clutch transmits 37 kW at 2000 rpm. (5)

P.T.O.

- Q6a) Derive the relation $T_1/T_2 = e^{\mu\theta}$ for flat belt drive where T_1 =Tight side tension, T_2 =Slack side tension, μ =coefficient of friction between belt and pulley and θ = angle of contact of belt with the pulley. (5)
- b) A V-belt of 6 cm² cross-section has a groove angle of 40° and an angle of lap of 150°, $\mu=0.10$. The mass of belt per metre run is 1.2 kg. The maximum allowable stress in the belt is 850 N/cm². Calculate the horse power that can be transmitted at a belt speed of 30m/s. (3)
- c) Classify various types of chains. (2)
- Q7a) Sketch and explain the working of Bevis Gibson flash light torsion dynamometer. (3)
- b) A simple band brake is operated by a lever of length 500 mm long. The brake drum has a diameter of 500 mm and the brake band embraces 5/8 of the circumference. One end of the band is attached to the fulcrum of the lever while the other is attached to a pin on the lever 100 mm from the fulcrum. If the effort applied to the end of the lever is 2000 N and the coefficient of friction is 0.25, find the maximum braking torque on the drum. (4)
- c) Define the following in relation to governors: A) Sensitivity
B) Stability
C) Isochronism (3)

X-X-X